THE UNSTABLE PLANE MOTION OF THE POISEUILLE TYPE FOR RIVLIN- ERIKSEN FLUID*

A. GEORGESKU and S.S. CHETTI

Exact unsteady solutions are sought, which describe a plane flow of the Poiseuille type for the Rivlin-Eriksen fluids for arbitrary pressure drop. All considered here flows become, after infinitely long times, a plane steady Poiseuille flow. In particular cases the results obtained here coincide with those obtained earlier in /1/. Certain statements formulated in /1/ without proof are strictly proved here.

The unsteady plane flow of the Poiseuille type for Rivlin-Eriksen fluid between planes $y = \pm 1$ is defined by the equations with initial and boundary conditions

$$\frac{\partial u}{\partial t} = -\frac{\partial p}{\partial x} + \frac{1}{R} \frac{\partial u^2}{\partial y^2} + S \frac{\partial^2}{\partial y^2} \left(\frac{\partial u}{\partial t}\right),$$

$$0 = -\frac{\partial p}{\partial y} + 2 \left(2S + S_1\right) \frac{\partial u}{\partial y} \frac{\partial u^2}{\partial y^2}$$
(1)

$$\begin{array}{l} u(y,t) = 0; \quad -1 \leqslant y \leqslant 1, \ t \leqslant 0; \\ u(y,t) = 0, \ t > 0, \ y = \pm 1 \end{array}$$
(2)

where (u(y, t), 0) is the dimensionless velocity, p = p(x, y, t) is the pressure, R > 0 is the Reynolds number, S > 0 is the parameter of viscoelasticity, and $uS_1 > 0$ is the lateral velocity parameter. From the second of Eqs. (1) follows that $\partial p/\partial y$ is a function of only y and t and, consequently, $p = f_1(y, t) x + f(y, t)$. Substituting this expression into the first of Eqs. (1), we stipulate $f_1(y, t) = -f(t)$, hence p = -f(t) x + g(y, t) when x is the axis parallel to the walls. On the assumption that function f(t) is given, we define velocity u by the first of Eqs.

(1) and conditions (2). After that pressure $\partial p/\partial y$ is determined by the second of Eqs.(1). The expression for p implies that the flow must be considered in a bounded region. Since

the drop of controlling pressure depends on the motion of fluid, it is reasonable to determine it as in classic Newtonian case.

The velocity u satisfies conditions (2) and the first of Eqs.(1) in which $\frac{\partial p}{\partial x} = f(t)$. Restricting the analysis to the consideration of velocities with exponential damping as $t \to \infty$. by the method of Laplace transformation, we obtain

$$u(y, t) = \int_{0}^{t} f(\tau) H(y, t-\tau) d\tau, \quad t \in [0, \infty], \quad y \in [-1, 1]$$

$$H(y, t) = \sum_{n=0}^{\infty} \frac{4 (-1)^{n} (1+\lambda_{n} RS)}{\pi (2n+1)} \cos \left[\frac{\pi}{2} (2n+1) y \right] \exp(\lambda_{n} t)$$

$$\lambda_{n} = -\frac{\pi^{2} (2n+1)^{2}}{4R + \pi^{2} (2n+1)^{2} RS}$$
(3)

Since $0 < 1 + RS\lambda_n < 1$, $\lambda_n < 0$, the series for H(y, t) is uniformly convergent. Hence it is possible in formula (3) to exchange the signs of the integral and sum. Note that the uniform convergence not only of the series determining H but, also, of other series that occur below, can be proved on the basis of properties of λ_n .

From the theory of Laplace transformations we have

$$\lim_{t\to\infty} u(y, t) = \lim_{\pi\to 0} \lambda \bar{u}(y, \lambda) = \lim_{\lambda\to 0} \left[\lambda \left(\overline{f(\lambda)}\right)\right] \frac{R}{2} \left(1 - y^2\right) = \frac{R}{2} \left(1 - y^2\right) f(\infty)$$

From this follows that all obtained here solutions approach the Poiseuille classic solution.

*Prikl.Matem.Mekhan.,Vol.47,No.2.pp.342-344,1983

It is known that in the case of plane steady Poiseuille flow for Newtonian fluids it is possible to specify either the pressure drop $\partial p/\partial x = -f(\infty)$ or the friction $(\partial u/\partial y)_{y=\pm 1} = \mp Rf(\infty)$ at the walls.

Similarly in the Newtonian case u(y, t) is uniquely determined by either $\partial p/\partial x$ or $(\partial u/\partial y)_{y=\pm 1}$ From formula (3) we obtain

$$\left(\frac{\partial u}{\partial y}\right)_{y=\pm 1} = \pm \sum_{n=0}^{\infty} 2\left(1 + \lambda_n RS\right) \int_0^t f(\tau) \exp\left[\lambda_n \left(t - \tau\right)\right] d\tau = \pm 2 \int_0^t f(\tau) M\left(t - \tau\right) d\tau$$
$$M\left(t - \tau\right) > 0$$

For a specified friction at the walls the obtained results is an equation of the Volterra. type, where f is a known function. As $M(t-\tau) > 0$ for any t > 0, $(\partial u/\partial y)_{y=\pm 1} = 0$ for t = 0, while M and $(\partial u/\partial y)_{y=\pm 1}$ are differentiable functions with respect to t in any interval (0, a), a > 0, we conclude, in conformity with the Volterra theorem, that each of these equations has a unique solution. It is clear that for a given function f(t) we have a unique functions defining friction at the wall.

Let us now consider the particular case of $f(t) = P(1 - \exp(-\omega t))$, $P, \omega > 0$, in which solution (3) becomes the solution given in /l/. Representing function u in the form of suitable Fourier series, we obtain the following formula:

$$u(y, t) = \frac{P}{\omega} \left[1 + \frac{\cos hy}{\cos h} \right] (\exp(-\omega t) - 1) +$$

$$\sum_{n=0}^{\infty} 16 (-1)^{n-1} \frac{PR\omega \cos[\pi (n + 1/2) y]}{(2n+1)^3 \pi^3 (\omega + \lambda_n)} (\exp(\lambda_n t) - 1)$$

$$h = \sqrt{\frac{R\omega}{1 - RS\omega}}, \quad y \in [-1, 1], \quad t \in [0, \infty)$$
(4)

To obtain expressions for derivatives of u it is necessary to investigate additionally their behavior at points $y = \pm 1$, where the respective Fourier series do not always converge to the expanded function.

To find the derivatives we used the technique of reverse integration in conformity with /2/. It follows from formula (4) that

$$\frac{\partial u}{\partial y} = \frac{P}{\omega} h \frac{\sin hy}{\cos h} (\exp(-\omega t) - 1) -$$

$$\sum_{n=0}^{\infty} 8 (-1)^{n-1} \frac{PR\omega \sin [(n+1/2) y]}{(2n+1)^2 \pi^2 (\omega + \lambda_n)} (\exp(\lambda_n t) - 1)$$

$$y \in [-1, 1], t \in [0, \infty)$$

$$p = -P(1 - \exp(-\omega t))x + (2S + S_1) \left(\frac{\partial u}{\partial y}\right)^2 + \text{const}$$
(5)

The last expression shows that pressure increases an account of Newtonian terms. Applying formula (3) to the particular cases considered in /3/, we again obtain all the results of /3/.

In the particular cases considered above the numerical computations were carried out for $P = 1, S = 1, 2, 5, 10, 100, R = 1, 4, 10, 100, \omega = 2, 10, 100$ and for different values of t and y.



The results of computations are shown in Fig.1 for P = 1, R = 1, $\omega = 10$ by solid lines for S = 1 and by dash lines when S = 100. Curves 1-3 correspond to t equal 0.3, 5, and 100. These curves and, also, formulas (3) show that when t=0 the profile of velocity is a straight line u=0, $y \in (-1, 1)$, for t > 0, u is a positive increasing function of t which as $t \to \infty$ approaches the Poiseuille parabolic profile. The profile of velocity is symmetric about the x axis. Furthermore u is an increasing function of P and independent of S_1 .

Pressure increases with S_1 . It will be seen that increasing R or ω and decrease of S results in increasing velocity.

The authors thank D. Chomentowski for discussion of some of the results and, also, V. Nilolae for help in computations.

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Translated by J.J.D.

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